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STRESS: A PROBLEM-ORIENTED LANGUAGE FOR STRUCTURAL ENGINEERING

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ABSTRACT:

STRESS is a general purpose programming system for the analysis of structures. Compared to other systems, STRESS has some distinguishing characteristics. The first is that of the structural engineer who uses STRESS. The second is the ease with which the engineer can use STRESS. The third is the ease with which a wide variety of structural problems can be solved. The fourth is the material use as a design tool; and the fifth is the ease with which the fact that modifications of the structure can be made and can be easily analyzed. The last of these characteristics is used in the design of structures. STRESS is used to generate a system which can analyze a structure and can also analyze but, more significantly, enables the engineer to realize an efficient structure.

INTRODUCTION

Presented in this paper is a brief description of STRESS (Structural Engineering Systems Solver) which is a system for structural analysis by digital computer. It consists of a language which describes the structural problem and a processor which produces the requested results. STRESS is a general purpose system in the sense that it is capable of analyzing a wide variety of structural types and situations. The input language is problem-oriented, i.e., the only problem description required is in engineering rather than computer language.

On the assumption that the reader is not a structural engineer, a few words concerning the general nature of the structural design problem appear to be in order. For example, consider the simple building frame shown in Fig. 1. The members of this structural system may be of steel, reinforced concrete or some other material and are rigidly connected at the joints. The objective of design is to evolve a structure which will support the imposed loads without excessive stress or deformation and with maximum economy.

The analysis of the relatively simple frame in Fig. 1 requires the determination of 63 distinct force and moment components. This is accomplished by the solution of an equal number of equations. Forty-two of these are classified as equilibrium equations. The remainder express the compatibility of distortions between the various elements. The total set of equations may be subdivided such that analysis requires the solution of 21 simultaneous equations. It should be apparent that rigorous analysis of a more sizable structure (e.g., a 20-story building frame) requires an enormous amount of computation and data processing.

The problem is further complicated by the fact that the deformation of the individual members and hence the compatibility equations depend upon the size and elastic properties of those members. Hence design must be an iterative process each cycle of which involves a new analysis of the complete structure and a revision of the member sizes.

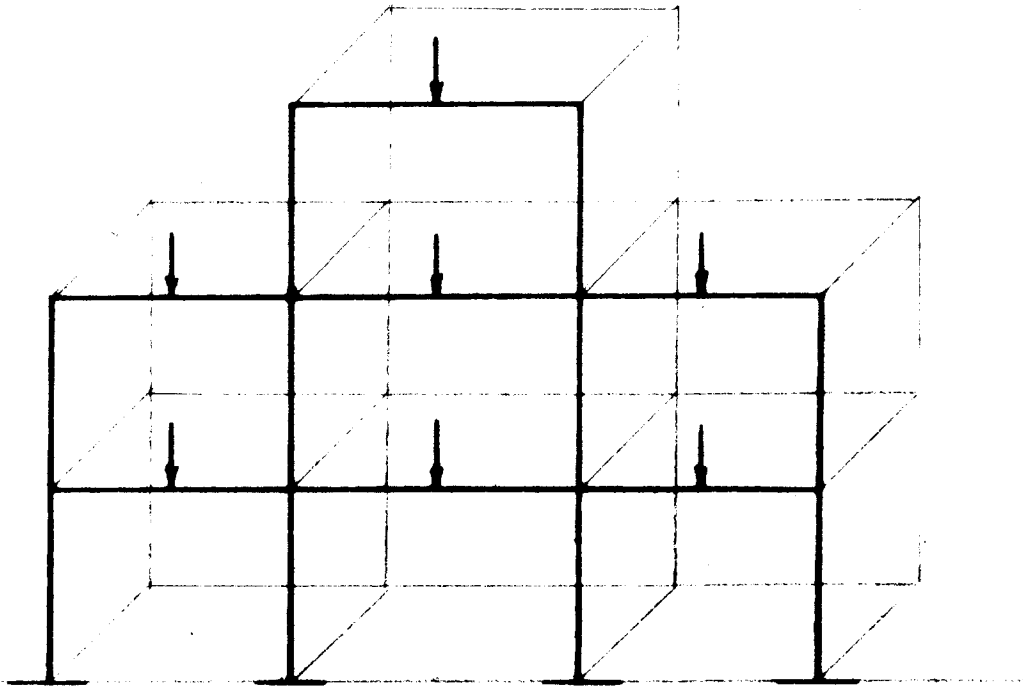


Fig. 1 Rigid Building Frame

Before the advent of computers rigorous or "exact" analysis was possible only in the most simple cases. For structures of moderate size relaxation techniques were commonly used. In the case of larger structures it was necessary to employ approximate methods. These latter methods were very ingenious and resulted in structures which were satisfactory from the viewpoint of safety and performance. However, because of the approximation in analysis a degree of conservatism was prudent and structures were seldom optimal with respect to economy. Furthermore, the very time-consuming computation involved, even with the approximate methods, made it impractical to conduct the repetitive analysis necessary to converge on an optimum design.

Because of the nature of the problem structural analysis is ideally suited to electronic computation. Structural engineers were quick to recognize this fact and extensive use has been made of computers over the past decade. This usage was primarily in the form of special purpose programs, i.e. programs written for the analysis of a particular structure. Since the time and money required to develop a new program is considerable this form of computer application is restricted to large, important structures and is not a satisfactory solution to the general structural engineering problem.

More general programs have been written but these are also restrictive in that they may be used only for a specific type of structure, e.g., continuous stringer bridges. Although a step in the right direction, this attack falls far short of the ultimate objective of making the computer readily accessible to the engineer for any purpose. Such programs are also undesirable because of the understandable tendency of the designer to make his structure fit the available program.

The most serious deficiency in the current mode of computer usage in structural engineering is the lack of direct communication between the engineer as such and the machine. The engineer has had two choices: He could become a programmer himself or he could turn the analysis over to a middleman who was a computer expert but probably did not fully

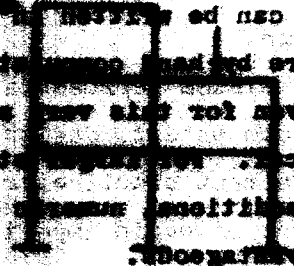
understand the structural problem. The first choice is impractical because the broader aspects of design fully tax the engineer's capacity leaving little time for diversion into another field. The second choice is undesirable because the engineer loses control of his own design.

STRESS represents an attempt to automate structural analysis and, to some extent, design in such a way that some of the difficulties in current practice are alleviated. The system has three characteristics of basic importance which distinguish it from previous efforts: (1) The only input required is in engineering rather than machine language thus making possible full use of the system by an engineer not trained in computer programming; (2) It is a general purpose program capable of handling structures of many types which comprise the majority of analysis problems encountered in structural engineering; and (3) Modifications of the original structure may be easily made thus expediting the iterative design process. The latter capability is most effective when STRESS is used in the time-sharing mode which permits an engineer to actually design a structure while sitting at a console. Design is a decision-making process and the role of STRESS in the time-sharing mode is to immediately process the data on which decisions are based.

The types of structures which can be analyzed by STRESS include those shown in Fig. 2. The structure may be either two or three dimensional and the joints may be considered pinned as in a riveted truss or rigid as in a welded steel frame. The members may be prismatic or may vary in cross-section along the length. The loading may be in the form of concentrated or distributed forces and moments or may be the result of temperature changes or support movement. This generality of application is unique since the structures shown in Fig. 2 are usually considered to be of distinct types requiring different methods of analysis.

to a mathematician who was a computer expert and probably did not know

A typical STRESS problem description is shown in Fig. 3. Although the example is trivial it serves to demonstrate the simplicity of the STRESS input language. The input, which is completely shown in the figure, can be written in a matter of minutes. An analysis of this structure for the case shown would require approximately one hour. Thus, even for the more complex cases, the use of a computer becomes economical. The input structure the input program is entered only by the additional numerical data required and the output is more advantageous.



The important point to be made in connection with Fig. 3 is that the program consists of programming terms such as "member", "node", etc. An engineer trained in structural analysis can learn the STRESS language in a few hours. It is then in a position to analyze by computer the majority of structures which he encounters in practice. In other words, the engineer who will make the design decisions is in direct communication with the machine on his own terms. By this means the use of computers in structural engineering becomes economical, not only for the large, complex problems as at present, but for the routine day-to-day analysis which comprises the bulk of professional practice.

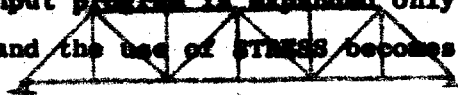


Integration of Structural Analysis

The method of matrix analysis has recently been applied to the solution of the problem of the linear structure problem. (1) It is shown that the electrical network and the structural network. (2) It is formulated and solving the linear structural analysis problem using network theory. The method of matrix analysis used in this work is based on this work, although the method has more recently been derived more explicitly by Connor. (3)

The electrical network consists of branches and nodes, with analysis quantities being associated with the properties and variables. The structural analogy to the branch is the member, the joint to the node. The unknown at a point in a member is not a scalar, but a vector. The

A typical STRESS problem description is shown in Fig. 3. Although the example is trivial it serves to demonstrate the simplicity of the STRESS input language. The input, which is completely shown in the figure, can be written in a matter of minutes. An analysis of this structure by hand computation would require approximately one hour. Thus, even for this very simple case, the use of a computer becomes economical. For larger structures the input program is expanded only by the additional numeric data required and the use of STRESS becomes more advantageous.



The important point to be made in connection with Fig. 3 is that the program consists of engineering terms such as "frame", "joint", "member", etc. An engineer trained in structural analysis can learn the STRESS language in a few hours. He is then in a position to analyze by computer the majority of structures which he encounters in practice. In other words, the engineer who will make the design decisions is in direct communication with the machine on his own terms. By this means the use of computers in structural engineering becomes economical, not only for the large, complex problems as at present, but for the routine, day-to-day analysis which comprises the bulk of professional practice.

Formulation of Structural Analysis

A great deal of interest has recently been shown in the application of network theory to the framed structures problem. Branin⁽¹⁾ showed the analogies between the electrical network and the structural network. Fenves joined Branin⁽²⁾ in formulating and solving the linear structural analysis problem using network theory. The method of analysis used in STRESS is based on this work, although the method has more recently been derived more explicitly by Connor⁽³⁾.

The electrical network consists of branches and nodes, with scalar quantities being associated with the properties and variables. The structural analogy to the branch is the member, the joint to the node. The unknown at a point in a member is not a scalar, but a vector. The



joint variables also are vectors. The number unknown at another point in the member is related, not by a linear transformation, but by a

STRUCTURE PROPERTIES

TYPE PLANE FRAME
NUMBER OF JOINTS 5
NUMBER OF MEMBERS 4
NUMBER OF SUPPORTS 2
NUMBER OF LOADINGS 1
METHOD STIFFNESS

JOINT COORDINATES

1 0. 0. 5
2 0. 150.
3 450. 225.
4 500. 150.
5 500. 0. 5

MEMBER INCIDENCES

1 1 2
2 2 3
3 3 4
4 4 5

MEMBER PROPERTIES

1 AX 10. 17 1000
2 AX 10. 12 1500
3 AX 10. 12 1000
4 AX 10. 12 1000

LOADING WIND

JOINT 2 LOAD 1000

MEMBER 2 LOAD FORCE 1 UNIFORM

TABULATE FORCES REACTIONS DISPLACEMENTS

SOLVE

Considering a displacement or stiffness method of analysis, Table 1 also shows the minimum number of unknown vector components per joint for the analysis. For structural type analysis, the minimum number of equations per joint required for the space frame. By taking consistent axes the force and displacement components not shown in Table 1 are always zero and need not be considered. These zero values may be omitted from the vectors and the corresponding rows and columns deleted from the formations. Figure 5 shows schematically the deletion for a plane frame. Not only is the number of simultaneous equations necessary to solve held to a minimum, but almost all of the program is independent of structural type, related only by the joint displacement vector size.

joint variables also are vectors. The member unknown at another point in the member is related, not by a linear transformation, but by a matrix transformation. Figure 4 illustrates the force transformation from one end of a straight member to the other, with no forces applied in between. The general force vector for the three dimensional structure consists of three linear force components in an orthogonal system and three moment components acting about the axes. The general transformation matrix then has the size 6 x 6. Each column corresponds to the effect of a particular component on the right side of the equation, each row is used for the computation of one of the components on the left side. It can be shown that the displacement transformation is similar and equal to the transpose of the inverse of the force transformation.

The network concept allows us to readily deal conceptually with vectors of different sizes for different type of structures. The number of unknowns is then a function of the type, while the method of solution is not. When stated so simply this result may seem obvious but the fact is that for hand computation different methods have been used for different structure types. As a result, many computer programs have been written for the linear analysis of framed structures, each treating only one of the types listed in Table 1.

Considering a displacement or stiffness method of analysis, Table 1 also shows the minimum number of unknown vector components per joint (JF). For structural types other than the space frame the equations for the analysis can easily be formed by reducing the six equations per joint required for the space frame. By taking consistent axes the force and displacement components not shown in Table 1 are always zero and need not be considered. These zero values may be omitted from the vectors and the corresponding rows and columns deleted from the transformations. Figure 5 shows schematically the deletions for a plane frame. Not only is the number of simultaneous equations necessary to solve held to a minimum, but almost all of the program is independent of structural type, related only by JF, the joint displacement vector size.

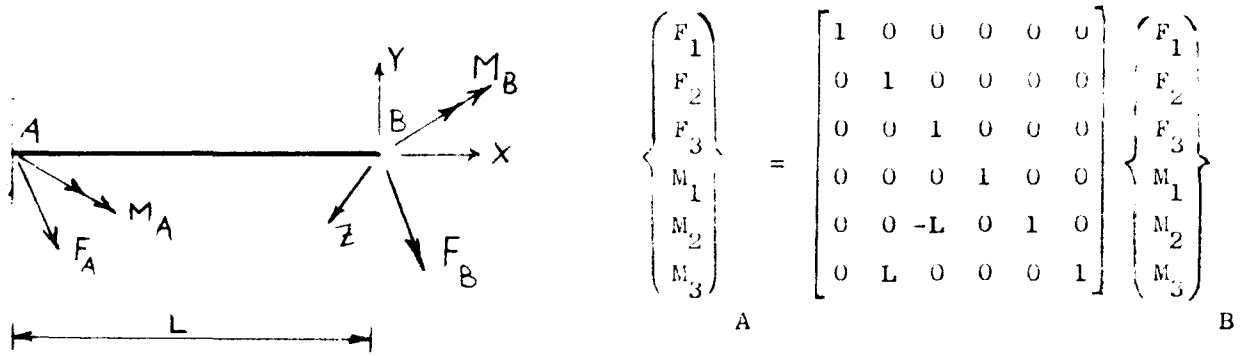


Fig. 4. Force Transformation for a Straight Member

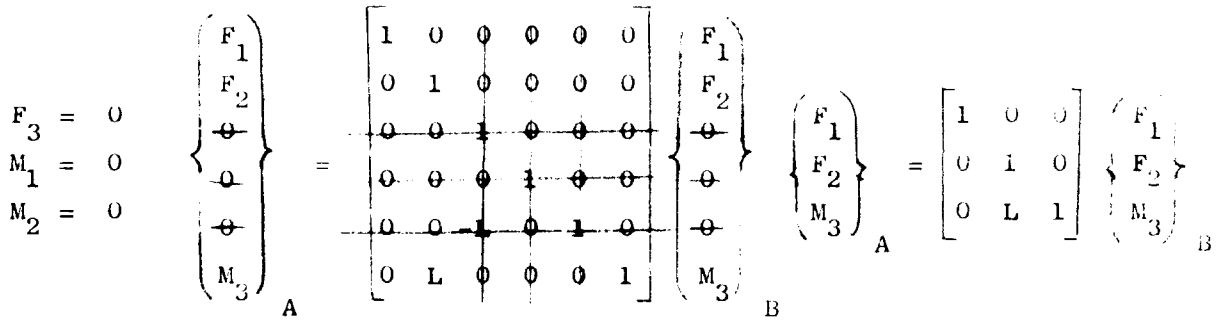
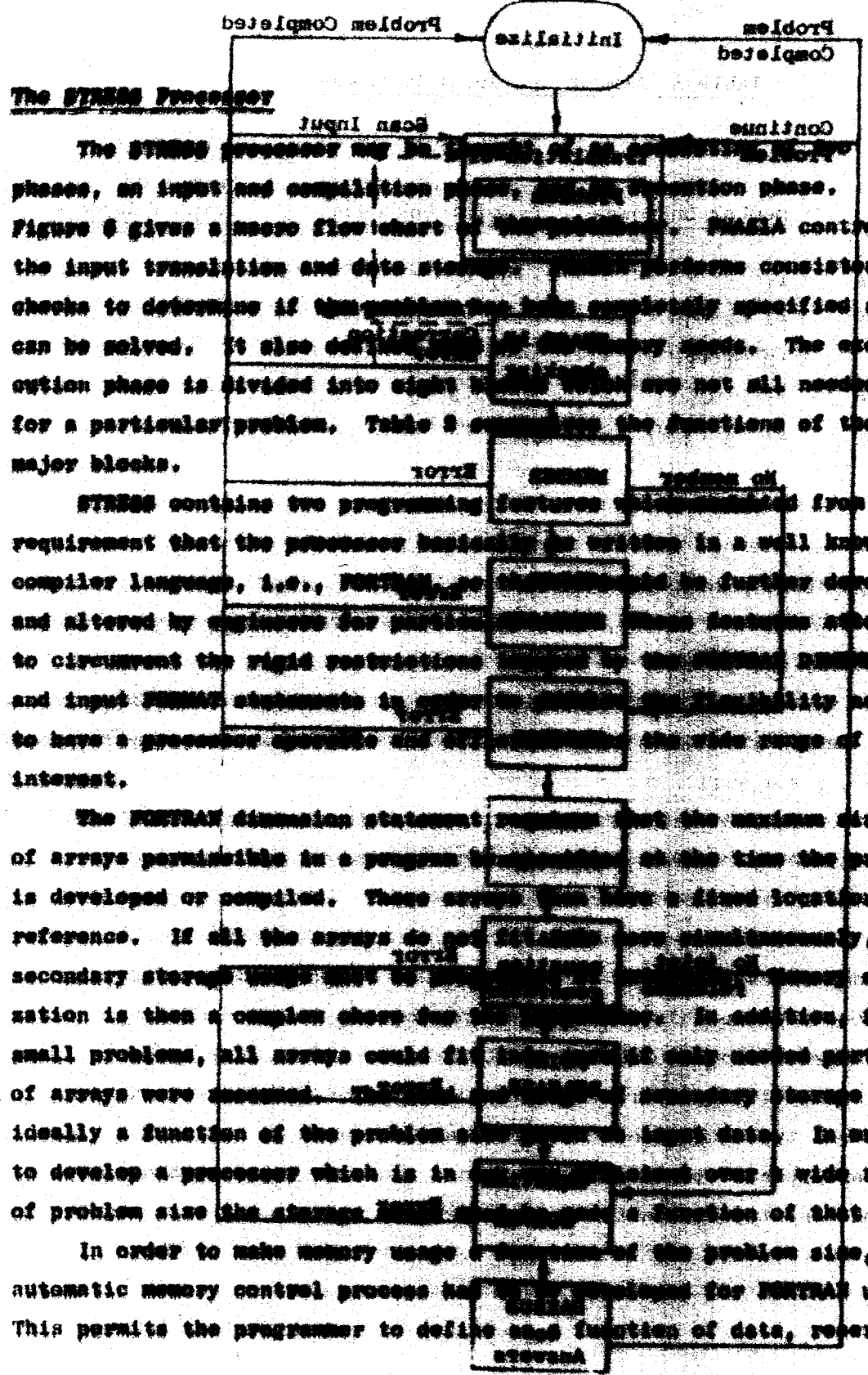


Fig. 5. Reduction of Transformation for a Plane Frame

Table 1 Member and Joint Unknowns

Type	Member Unknowns	Joint Unknowns	JF Number of Joint Unknowns
Plane Truss	F_1	$U_1 U_2$	2
Plane Frame	$F_1 F_2 M_3$	$U_1 U_2 U_3$	3
Grid	$F_3 M_1 M_2$	$U_3 U_4 U_5$	3
Space Truss	F_1	$U_1 U_2 U_3$	3
Space Frame	$F_1 F_2 F_3 M_1 M_2 M_3$	$U_1 U_2 U_3 U_4 U_5 U_6$	6

STRESS is intended to be an informative and easily usable structural design tool. The designer then must be able to specify his problem to the machine easily, rapidly and concisely. He should be able to specify the problem as he thinks of it, not in terms of how the machine solves it. He should be able to specify a problem without performing any computations during data preparation. This implies that the processor will deal with much more information than merely the generation and solution of the analysis equations. For example, the equations relate imbalanced joint forces which can be computed from a variety of load types considered by the designer. The machine will operate on joint coordinates while the designer might relate geometry to bays and stories, or spans. In the process of generating and solving the equations, and in this pre- and post-processing the machine must deal with a great amount of data, mostly in array form. The number of arrays and their sizes are variable functions of the input data, the structural type and size. The form and features of the STRESS processor are related to these problems and a desire that the processor be a dynamic entity expandable by engineers.



The STRESS Processor

The STRESS processor consists of three main phases, an input and compilation phase, a translation phase, and an execution phase. Figure 8 gives a more detailed flowchart of the STRESS processor. STRESS controls the input translation and data storage. STRESS also performs consistency checks to determine if the problem is solvable and if the problem can be solved. It also performs consistency checks. The execution phase is divided into eight major blocks. Table 2 summarizes the functions of these major blocks.

STRESS contains two programming features which are derived from the requirement that the programmer be able to write in a well known compiler language, i.e., FORTRAN. The first feature is that the programmer can alter and alter by engineers for particular problems. The second feature is that to circumvent the rigid restrictions imposed by the FORTRAN language and input FORTRAN statements is necessary. The flexibility needed to have a processor capable of handling a wide range of interest.

The FORTRAN dimension statement requires that the maximum size of arrays permissible in a program be specified at the time the program is developed or compiled. These arrays are then given a fixed location for reference. If all the arrays do not fit in memory simultaneously, secondary storage must be used. The organization of secondary storage is then a complex chess for the programmer. In addition, for small problems, all arrays could fit in memory. If only certain portions of arrays were accessed, secondary storage is ideally a function of the problem size and input data. In order to develop a processor which is in a wide range of problem size the storage must be a function of that size.

In order to make memory usage a function of the problem size, an automatic memory control process has been developed for FORTRAN usage. This permits the programmer to define the function of data, reserve

Figure 8. System Block Flow Chart

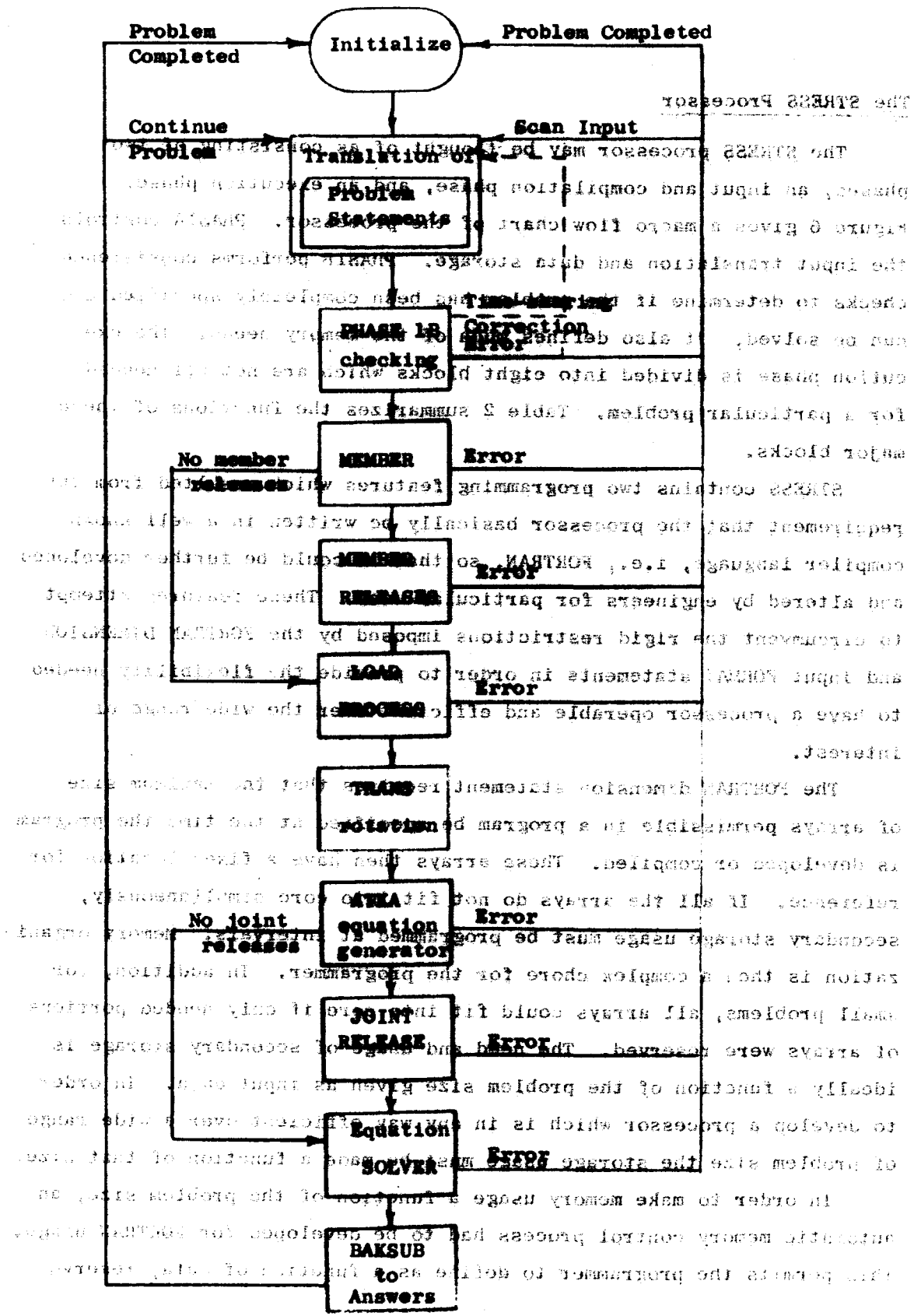


Figure 6. System Block Flow Chart

Table 2
Program Blocks

<u>NAME</u>	<u>PROCESS</u>
PHASE 1A	Translation
PHASE 1B	Consistency check, Internal representation
MEMBER	Compute member stiffness matrices
MRELES	Modify stiffness matrices for member end releases
LOAD PROCESSOR	Process all types of raw load data into equivalent joint loads
TRANS	Rotate member stiffness matrices into global coordinates
ATKA	Generate symbolically structural stiffness matrix
JRELES	Modify stiffness matrix and joint loads for joint releases
SOLVER	Solve, matrix equation for joint displacements
BAKSUB	Backsubstitute for other results and print.

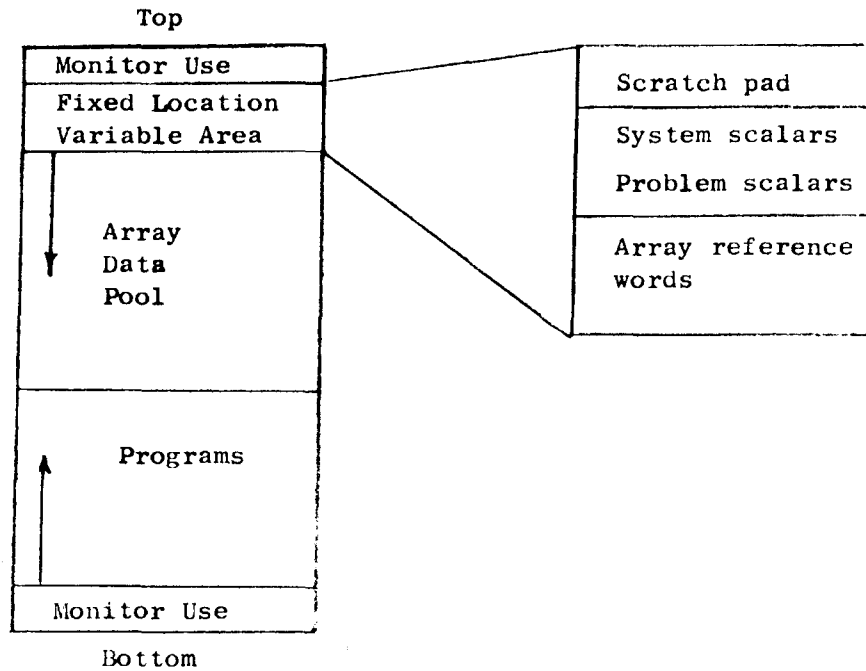


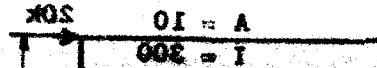
Figure 7. Core Memory Layout

space for, reference and use the arrays without requiring the location of the array to be fixed or even constant during a part of the solutions process. Figure 7 shows the normal core arrangement for STRESS operation. A very small area (about 300 words) of fixed location variables is included in upper core. Part of this area contains codewords which are used to reference the arrays. These codewords contain such information as the array size and location, and or out of core. All of the pertinent information can be changed by program control. The remaining memory down to the top of program consists of a pool for arrays. When the pool is full, it is reorganized, using secondary storage.

The amount of program comprising the system has exceeded core capacity. Program blocks must then be swapped during processing. The FORTRAN chain feature, with modifications, is used for this purpose. With each program block there is a different top of programs, or bottom of the pool. This results in a constantly varying data memory capacity, easily accounted for by the memory controller. A slightly different form of the memory is used with time-sharing, but this is conceptually no different.

The use of explicit FORMAT statements requires that a programmer know the form of an input card or line before recognizing the first character. In addition very rigid restrictions are placed on character position. This is inconsistent both with the rest of FORTRAN, which allows great freedom in source program format and elegant output, and the engineers scope of concern. It is necessary to provide the engineer with a free and easy form of input, free field format and great freedom in statement ordering.

A single small subroutine was written to do operations on logical (rather than physical) input fields, performing dictionary look-up, binary conversion, etc. This routine is called for every logical data field during translation of input data by translation programs written in FORTRAN. The programmer then has the input capabilities usually found only in a compiler or other extensive assembly language programs.



The input programs are then available to a computer language compiler. These programs are then compiled to instructions and computer restrictions and are available to the user of the computer languages for development of dynamic engineering systems is unquestionably great. We only regret the situation of the computer languages in the new generation computers.

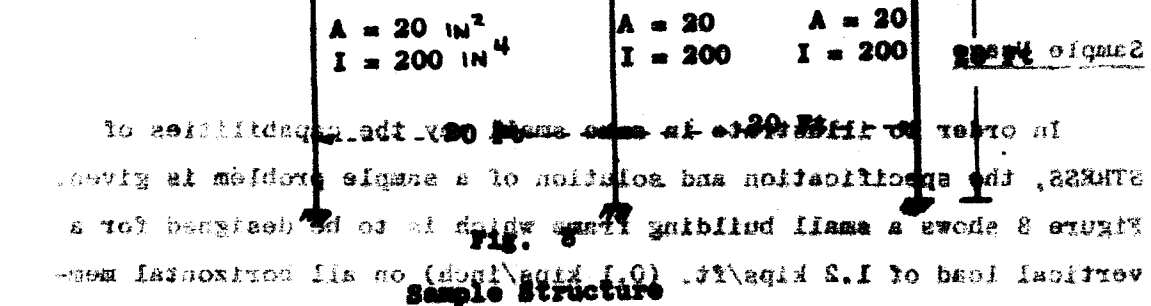
Sample Name

OS = A OE = A ² or OS = A
 OOS = I OOE = I ³ or OOE = I

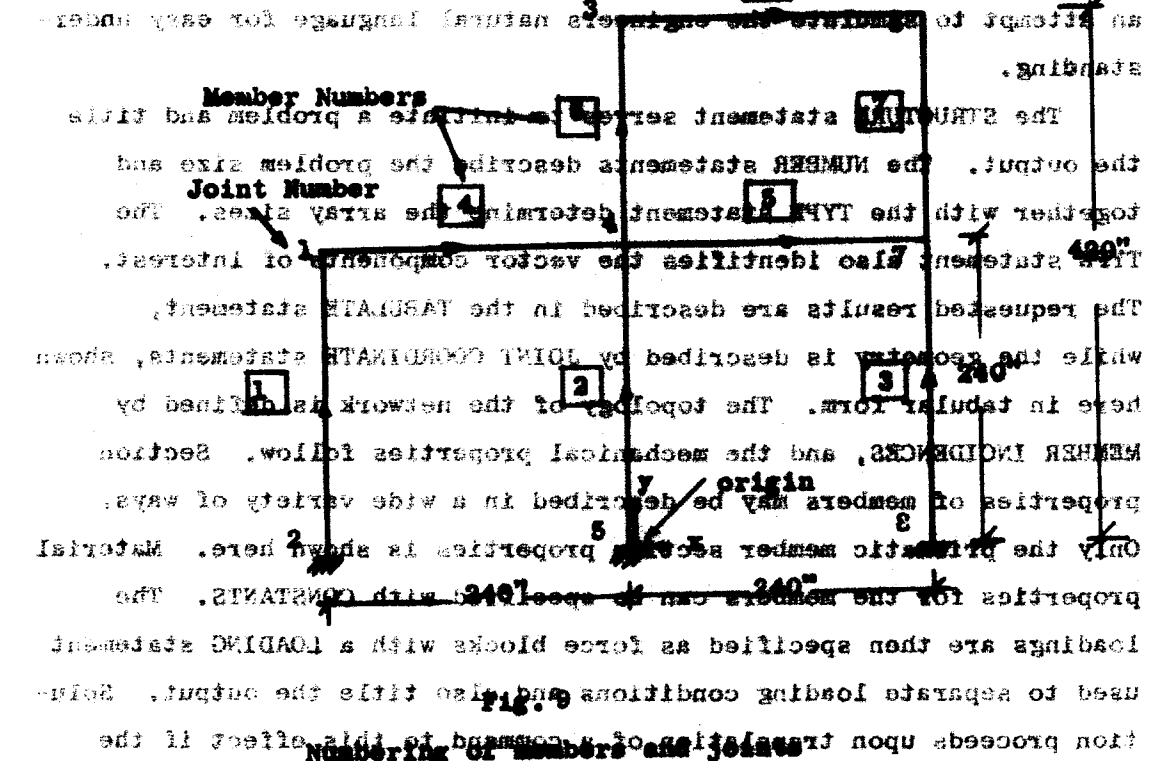
In order to illustrate the use of the computer languages of STRESS, the specification and solution of a simple problem is given. Figure 8 shows a small building designed to be designed for a vertical load of 1.5 kips/ft. The building is 10 ft high and 10 ft wide and a 20 kips per ft horizontal load. Figure 9 shows the computer oriented representation of the structure with location numbering of joints and members, and defining the member orientations. Units are also made consistent with the computer language, which is an attempt to provide a consistent language for easy understanding.

The STRESS statement and the output and title the output. The STRESS statement contains the problem data and together with the TITLE statement, the STRESS output. The TITLE statement has the following information of interest. The requested results are described in the STRESS statement, while the procedure is described in the STRESS statements, shown here in typical form. The material properties are defined by MEMBER PROPERTIES, and the member orientations are defined by MEMBER ORIENTATIONS. The material properties are defined in a similar way of ways. Only the material properties are defined in a similar way. Material properties for the members are defined in a similar way. The loadings are then specified in a similar way. The STRESS statement used to separate loading conditions is given in the output. Solution proceeds upon translation of the program to the effect of the

The input programs are then analyzed to a computer language compiler. The programs are then analyzed to illustrate some computer results. The programs are mentioned in the value of the computer languages. The value of the computer languages is mentioned in the value of the computer languages. We only refer to the value of the computer languages in the next generation compilers.



In order to solve the problem, the specification and solution of a sample problem is given. Figure 8 shows a small building frame which is to be designed for a vertical load of 1.5 kips/ft. (0.1 kN/m) on all horizontal members and a 30 kips per floor wind loading. Figure 9 shows the computer oriented representation of the structure which involves a list of joints and members, and defining the member orientations. Table 3 shows the SIRSS input, which is also made constant.



Only the plastic member properties are shown here. Material properties of members may be defined in a wide variety of ways. MEMBER INVOICES, and the mechanical properties follow. Section here in tabular form. The topology of the network is defined by while the results are described by JOINT COORDINATE statements, shown. The requested results are described in the TABULATE statement. This statement also identifies the vector components of interest. Together with the TYPE statement determining the array sizes. The output. The NUMBER statement describes the problem size and the SIRSS statement series. An attempt to solve the problem is made and this is also made constant.

Table 3
Sample Problem Specification

00010 STRUCTURE SAMPLE PROBLEM
00020 NUMBER OF JOINTS 8
00030 NUMBER OF MEMBERS 8
00040 NUMBER OF SUPPORTS 3
00050 NUMBER OF LOADINGS 2
00060 TYPE PLANE FRAME
00070 METHOD STIFFNESS
00080 TABULATE FORCES, REACTIONS
00090 JOINT COORDINATES
00100 1 X -240. Y 240. FREE
00110 2 X -240. SUPPORT
00120 5 X 0. S
00130 8 X 240. S
00140 4 Y 240.
00150 7 X 240. Y 240.
00160 3 Y 420.
00170 6 X 240. Y 420.
00180 MEMBER INCIDENCES
00190 1 2 1
00200 2 5 4
00210 3 8 7
00220 4 1 4
00230 5 4 7
00240 6 4 3
00250 7 7 6
00260 8 3 6
00270 MEMBER PROPERTIES
00280 8 PRISMATIC AX 10. IZ 300.
00290 4 PRISMATIC AX 10. IZ 300.
00300 MEMBER PROPERTIES PRISMATIC
00310 1 AX 20. IZ 200.
00320 2 AX 20. IZ 200.
00330 3 AX 20. IZ 200.
00340 5 AX 10. IZ 300.
00350 6 IZ 180. AX 20.
00360 7 IZ 180. AX 20.
00370 CONSTANTS E 30000. ALL
00380 LOADING 1 UNIFORM ALL BEAMS
00390 MEMBER LOADS
00400 8 FORCE Y UNIFORM -0.1
00410 4 FORCE Y UNIFORM -0.1
00420 5 FORCE Y UNIFORM -0.1
00430 LOADING 2 WIND FROM RIGHT
00440 JOINT LOADS
00450 6 FORCE X -20.
00460 7 FORCE X -20.
00480 SOLVE THIS PART

statements prior to this command constitute a complete and consistent problem.

For efficient use of time-sharing, the input is typed in using the CTSS monitor input program in a form which STRESS can accept and execute. The remote console is used for controlling the processor and for immediate correction of errors so as not to delay the design. Answers to the specified problem are shown in Table 4.

The results show the forces acting on the member ends and acting on the joints. With the solution of the member end forces, the member is statically determinate, so that the forces and deformations in the interior of the member can be determined by elementary methods. Up to now the development of STRESS has concentrated on the overall problem. We are now, however, attacking such problems as the interior forces to develop a more effective design aid. The joint loads on support joints represent the reactions. While the difference between the calculated joint loads and the applied joint loads gives a measure of the solution accuracy.

The engineer may then wish to alter the problem for his developing design. In most cases the alterations will be a function of the obtained results which were not known during creation of the input file. He might then describe the differences in the new problem to the processor and obtain results for immediate comparison and evaluation of the merits of the tact of the design. Table 5 shows the changes necessary to analyze the same structure with new member properties as suggested by the first analysis. Table 6 shows the effects of the changes.

The STRESS system is in a continuing state of development. It is expected that its capability will be extended to include dynamic analysis, investigation of structural stability, and the behavior of inelastic structures. It is hoped that ultimately STRESS will become part of a larger system which will be an aid to automatic structural optimization.

Table 4
Sample Problem Results

STRUCTURE SAMPLE PROBLEM
LOADING 1 UNIFORM ALL BEAMS

MEMBER FORCES

MEMBER	JOINT	AXIAL FORCE	SHEAR FORCE	BENDING MOMENT
1	2	10.545	-1.229	-92.604
1	1	-10.545	1.229	-202.414
2	5	38.982	0.481	44.045
2	4	-38.982	-0.481	71.498
3	8	22.473	0.748	65.663
3	7	-22.473	-0.748	113.813
4	1	1.229	10.545	202.414
4	4	-1.229	13.455	-551.690
5	4	-1.846	13.366	628.394
5	7	1.846	10.634	-300.563
6	4	12.161	-2.594	-148.203
6	3	-12.161	2.594	-318.751
7	7	11.839	2.594	186.750
7	6	-11.839	-2.594	280.204
8	3	2.594	12.161	318.751
8	6	-2.594	11.839	-280.204

STRUCTURE SAMPLE PROBLEM
LOADING 1 UNIFORM ALL BEAMS

JOINT LOADS

JOINT	X FORCE	Y FORCE	MOMENT
SUPPORT REACTIONS			
2	1.2292	10.5447	-92.6044
5	-0.4814	38.9819	44.0448
8	-0.7478	22.4734	65.6634
APPLIED JOINT LOADS			
1	-0.0000	0.0000	-0.0000
3	0.0000	0.0000	0.0000
4	0.0000	-0.0000	-0.0000
6	-0.0000	0.0000	-0.0000
7	0.0000	0.0000	-0.0000

STRUCTURE SAMPLE PROBLEM
LOADING 2 WIND FROM RIGHT

MEMBER FORCES

MEMBER	JOINT	AXIAL FORCE	SHEAR FORCE	BENDING MOMENT
1	2	11.195	-13.334	-1776.267
1	1	-11.195	13.334	-1423.969
2	5	10.377	-14.732	-1890.313
2	4	-10.377	14.732	-1645.392
3	8	-21.573	-11.934	-1659.204
3	7	21.573	11.934	-1194.941
4	1	13.334	11.195	1423.969
4	4	-13.334	-11.195	1262.902
5	4	16.467	13.385	1434.174
5	7	-16.467	-13.385	1778.188
6	4	8.188	-11.599	-1051.684
6	3	-8.188	11.599	-1036.215
7	7	-8.188	-8.401	-583.247
7	6	8.188	8.401	-928.867
8	3	11.600	8.188	1036.215
8	6	-11.600	-8.188	928.867

STRUCTURE SAMPLE PROBLEM
LOADING 2 WIND FROM RIGHT

JOINT LOADS

JOINT	X FORCE	Y FORCE	MOMENT
SUPPORT REACTIONS			
2	13.3343	11.1953	-1776.2675
5	14.7321	10.3774	-1890.3127
8	11.9339	-21.5727	-1659.2038
APPLIED JOINT LOADS			
1	0.0000	-0.0000	-0.0000
3	0.0001	0.	-0.0000
4	-0.0002	0.0000	-0.0000
6	-20.0001	-0.0000	-0.0000
7	-20.0001	0.0000	-0.0000

PART 1 OF PROBLEM COMPLETED.

Table 5
Modification Specifications

STRESS IS READY FOR INPUT.
TYPE
modification of first part - second cycle for member sizes
TYPE
changes
TYPE
member properties prismatic
TYPE
1 iz 800.6
TYPE
2 iz 889.9
TYPE
3 iz 800.6
TYPE
4 iz 583.3
TYPE
5 iz 800.6
TYPE
6 iz 446.3
TYPE
7 iz 339.2
TYPE
8 iz 446.3
TYPE
solve
PROBLEM CORRECTLY SPECIFIED. SOLUTION WILL PROCEED.

Table 6
Modification Results

STRUCTURE SAMPLE PROBLEM
MODIFICATION OF FIRST PART - SECOND CYCLE FOR MEMBER SIZES
LOADING 1 UNIFORM ALL BEAMS

MEMBER FORCES

MEMBER	JOINT	AXIAL FORCE	SHEAR FORCE	BENDING MOMENT
1	2	10.982	-1.729	-127.062
1	1	-10.982	1.729	-287.964
2	5	38.177	0.712	68.235
2	4	-38.177	-0.712	102.705
3	8	22.841	1.017	92.710
3	7	-22.841	-1.017	151.376
4	1	1.729	10.982	287.964
4	4	-1.729	13.018	-532.213
5	4	-1.831	12.993	585.341
5	7	1.831	11.007	-347.003
6	4	12.166	-2.848	-155.832
6	3	-12.166	2.848	-356.873
7	7	11.834	2.848	195.627
7	6	-11.834	-2.848	317.079
8	3	2.848	12.166	356.873
8	6	-2.848	11.834	-317.079

STRUCTURE SAMPLE PROBLEM
MODIFICATION OF FIRST PART - SECOND CYCLE FOR MEMBER SIZES
LOADING 1 UNIFORM ALL BEAMS

JOINT LOADS

JOINT	X FORCE	Y FORCE	MOMENT
SUPPORT REACTIONS			
2	1.7293	10.9823	-127.0625
5	-0.7123	38.1766	68.2353
8	-1.0170	22.8411	92.7096
APPLIED JOINT LOADS			
1	-0.0000	0.0000	0.0000
3	0.0000	0.0000	0.0000
4	-0.0000	-0.0000	0.
6	-0.0000	0.0000	-0.0000
7	0.0000	0.0000	-0.0000

STRUCTURE SAMPLE PROBLEM
MODIFICATION OF FIRST PART - SECOND CYCLE FOR MEMBER SIZES
LOADING 2 WIND FROM RIGHT

MEMBER FORCES

MEMBER	JOINT	AXIAL FORCE	SHEAR FORCE	BENDING MOMENT
1	2	9.680	-12.474	-1790.689
1	1	-9.680	12.474	-1203.048
2	5	11.910	-15.680	-2144.816
2	4	-11.910	15.680	-1618.268
3	8	-21.589	-11.847	-1760.015
3	7	21.589	11.847	-1083.169
4	1	12.474	9.680	1203.047
4	4	-12.474	-9.680	1120.039
5	4	16.769	14.103	1590.743
5	7	-16.769	-14.103	1793.879
6	4	7.487	-11.384	-1092.514
6	3	-7.487	11.384	-956.610
7	7	-7.487	-8.616	-710.710
7	6	7.487	8.616	-840.170
8	3	11.384	7.487	956.610
8	6	-11.384	-7.487	840.170

STRUCTURE SAMPLE PROBLEM
MODIFICATION OF FIRST PART - SECOND CYCLE FOR MEMBER SIZES
LOADING 2 WIND FROM RIGHT

JOINT LOADS

JOINT	X FORCE	Y FORCE	MOMENT
SUPPORT REACTIONS			
2	12.4739	9.6795	-1790.6894
5	15.6795	11.9096	-2144.8164
8	11.8466	-21.5892	-1760.0153
APPLIED JOINT LOADS			
1	0.0000	0.0000	-0.0000
3	0.0000	0.0000	0.0000
4	0.0000	-0.0000	0.0000
6	-20.0000	-0.0000	0.0000
7	-20.0000	0.0000	-0.0000

PROBLEM COMPLETED.

The development reported herein is the work of a group within the Civil Engineering Department at M.I.T. Special credit is due Prof. S. J. Fenves of the University of Illinois who was a visiting member of the M.I.T. faculty during the year 1962-63 and was largely responsible for the initial concept.

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References:

1. "Machine Analysis of Networks and Its Applications," by F. H. Branin, Jr., Technical Report 855, IBM Development Laboratory, Poughkeepsie, New York.
2. "A Network-Topological Formulation of Structural Analysis," S. J. Fenves and F. H. Branin, Jr., Journal of the Structures Division, A.S.C.E., August, 1963.
3. "Notes on Matrix Analysis," J. Connor, Civil Engineering Department, M.I.T., Fall 1963. (Unpublished)
4. "STRESS: A User's Manual," S. J. Fenves, R. D. Logcher, S. P. Mauch, and K. F. Reinschmidt, M.I.T. Press, Cambridge, Massachusetts, March 1964.