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CONTINUED DEVELOPMENT OF THE QUANDRY CODE by A.F. Henry, Philippe Finck, Han-Sem Joo, Hussein Khalil, Kent Parsons, Jos6 Perez

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### CONTINUED DEVELOPMENT OF THE QUANDRY **CODE**

*by*

A.F. Henry Philippe Finck Han-Sem Joo Hussein Khalil Kent Parsons Jose Perez

Energy Laboratory and Department of Nuclear Engineering

Massachusetts Institute of Technology Cambridge, Massachusetts 02139

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### Final Report on Continued Development of the QUANDRY Code

### **I.** Introduction

This report is a summary of the continued development and testing of the nodal code QUANDRY as applied to the analysis of detailed power distributions throughout the lifetime of light water moderated power reactors. The project has been supported by four utilities, Consolidated Edison Co., Northeast Utilities Services Co., Pacific Gas and Electric Co., and Public Service Electric and Gas Research Corp. The duration of the project was originally to be from September **1982** to September **1983.** However, in order to provide continued support for students during the **1983** Fall term and to get into phase with utility budgeting schedules, charges for faculty supervision were reduced and a no-cost extension for the period October **1983** through December **1983** was requested and agreed to.

The general goal of the work carried out has been to develop and test a method for predicting detailed power history throughout the lifetime of a light water moderated power reactor. We have based the scheme on the two-group nodal code QUANDRY, and the present work has been concerned with testing and refining that code so that it can be used to predict accurate values for local fuel-pin power throughout reactor lifetime in a straightforward, economical manner. The study has been concerned with three tasks:

- **1.** Testing in three dimensions.
- 2. Reconstruction of detailed flux shapes for three-dimensional nodes.
- **3.** The determination of albedoe boundary conditions.

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The completion of these tasks has led us to the conclusion that a production code embodying the QUANDRY scheme would be more accurate, more reliable, easier to run, and ultimately cheaper than the present nodal methods used by utilities.

#### II. Review of the Theory

Before presenting any results it will be helpful to review the theory, that underlies QUANDRY.

The basic unknowns in QUANDRY are the two-group volumeaveraged nodal fluxes. Node-face-averaged currents and fluxes also appear in the basic equations. For the ideal case of reactors composed of nuclearly homogeneous nodes, QUANDRY predicts the node-averaged fluxes very accurately (maximum error  $\sim 2\%$  for nodes 20 cm on a side). However, real LWR's are not nuclearly homogeneous, and the problem of finding "homogenized" cross sections that will reproduce the correct node-integrated reactor rates is a severe one. In fact, it can be shown that, if the conventional diffusion theory group-parameters and continuity conditions are employed, there is no set of homogenized cross sections that will reproduce reference results. QUANDRY circumvents this difficulty by permitting the face-averaged fluxes to be discontinuous across nodal interfaces. To accomplish this we have introduced "discontinuity factors" into QUANDRY. If correct values of these factors are provided, QUANDRY will reproduce exactly the nodal reaction rates of the standard model. The discontinuity factors are defined as the ratio of the true face-averaged group-flux for the node in the heterogeneous reactor to its value predicted by the basic nodal equations (the so-called "nodal coupling equations") when node-averaged group parameters and exact values of node-averaged fluxes, reaction rates, and leakages are

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used in those equations. It follows that, if correct discontinuity factors are used, the node-averaged fluxes, reaction rates, and the eigenvalue predicted **by QUANDRY** will be exact. Moreover, it is then possible to regenerate exactly face-averaged fluxes for the heterogeneous nodes making up the reactor. These latter quantities can be used to obtain approximate values of node corner-point fluxes **by** an interpolation method. Knowledge of face-averaged and corner-point fluxes then permits reconstruction of detailed flux shapes throughout a heterogeneous node.

Determination of "exact" discontinuity factors requires knowing the "exact" solution. Thus, in practice we must approximate. **A** basic ground rule which we have imposed on our approximate procedures for finding discontinuity factors and homogenized group parameters is to avoid quarter-core solutions and to obtain fine-mesh solutions with reactor heterogeneities represented explicitly only for regions the size of a single assembly, or, at most, a cluster of several assemblies. In addition, as is customary, we perform such calculations for only two-dimensional slices of an assembly. Since, for a given average temperature and density, the number of such slices that differ from one another at the beginning of life is significantly smaller than the number of assemblies in a quarter of the reactor, computing costs are reduced over what they would be if fine-mesh calculations for full quarter-cores were performed to obtain homogenized parameters.

Ordinarily this computational advantage would be lost as fuel depletion takes place since, because of differing flux gradients, the same type fuel assemblies in different core locations will deplete in different ways. However, for our test cases, we have been able to account for

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such differences while still performing assembly depletions based on zero-net-current boundary conditions. If further testing shows this procedure to be generally valid, the computational advantage will persist throughout reactor lifetime.

The approximate methods which we have developed for determining discontinuity factors and for reconstructing fuel pin powers have been described in our progress report for the period September 1982 to April **1983.** Rather than repeat this material we shall instead summarize an overall **QUANDRY** approach (yet to be implemented in a production program) for analysing criticality, nodal and pin power distributions throughout LWR lifetimes. The method involves several steps:

**(1)** Run LEOPARD, CASMO, EPRI-CELL, CPM, or some equivalent, in the usual way to determine cross sections for all the heterogeneous zones making up a fuel assembly.

(2) Use these parameters to run two-group, **2D,** heterogeneous, **PDQ** depletion problems based on zero-current boundary conditions for assemblies or color sets. Edit input for the usual HARMONY tables from these results, but also edit face-averaged currents and fluxes across the surfaces of the **QUANDRY** nodes that make up these regions (4 nodes per assembly for PWR's, **1** node per assembly for BWR's) and from these quantities determine discontinuity factors (to be added to the HARMONY tables). Also store the detailed fluxes from the assembly depletions.

**3.** Run **3D QUANDRY** depletion problems using depletion dependent homogenized nodal cross sections and radial discontinuity factors from the HARMONY tables. These problems may be run either with reflector regions represented explicitly or **by** use of albedoe boundary conditions (see section VI **).** Section.IV describes how the axial discontinuity

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factors (when they are not unity) can be found internal to **QUANDRY.**

4. If it is desired to reconstruct fuel pin powers, then nodecorner-point fluxes and average fluxes on line-segments between cornerpoints must be interpolated from the **QUANDRY** output (see section **V).** The actual reconstruction need only be carried out for the nodes of interest (usually high power nodes). For PWR's, fine-mesh assembly or color set flux shapes modulated **by** bi-quadratic "form functions" appear to yield hot fuel pin powers which match detailed quarter-core **PDQ** results throughout life to within a few percent. To achieve comparable accuracy for BWR's requires that the **QUANDRY** output be used in conjunction with response matrices or that it be applied to specify the boundary conditions for a fine mesh reconstruction involving the assembly for which the pin powers are desired and (at least some of) its nearest neighbors. The September **1982** to April **1983** progress report describes these procedures in more detail.

It should be noted that the **QUANDRY** approach requires that finemesh calculations be performed only for assemblies or color sets. No quarter core PDQ's are needed. On the other hand, if quarter core PDQ's are available, they can be used to edit discontinuity factors which when used in QUANDRY will reproduce exactly the **PDQ** node-integrated reaction rates.

### III. Testing in Three Dimensions

Although **QUANDRY** has been applied with impressive accuracy to three-dimensional problems with homogeneous nodes, its accuracy for three-dimensional LWR problems with both radial and axial heterogeneities had not previously been tested. The most direct method of solving problems with axial heterogeneities is to choose the z-direction node

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boundaries to coincide with material discontinuities. In this manner, each node is relatively homogeneous in the axial direction and, consequently, homogenization procedures may only be required in two dimensions (although possibly corrected for axial leakage effects). It is clearly desirable to avoid the performance of three-dimensional assembly calculations to generate nodal parameters.

Unfortunately, two-dimensional assembly shapes provide no systematic way of estimating the surface flux discontinuity factors at node boundaries in the z-direction. Hopefully, the choice of axial mesh boundaries at material discontinuities makes the assumption of homogen-. ized flux continuity at these boundaries a good approximation. Accordingly we have undertaken to test the QUANDRY nodal procedure with unity discontinuity factors at axial node boundaries.

The results of doing this for two small 3D problems, representative of inner portions of a PWR and a BWR, were presented in the September-April progress report. We shall merely summarize them here.

The PWR benchmark problem was designated CC3. A layout is shown on Fig. 1. The assemblies used (rodded or unrodded) are shown on Fig. 2. Two fuel enrichments (1 and 2) are present and one of the assemblies contains a partially inserted control rod (shaded area in Fig. 1, composed of 16 shaded regions - not the central one - of Fig. 2). The other assemblies (non-fuel composition - 3) contain 17 water holes. The flux boundary conditions are as indicated.

A 3D PDQ reference for this problem was provided by Northeast Utilities. Table 1 shows errors in nodal power as compared with this reference. All axial discontinuity factors were taken as unity and radial discontinuity factors were taken as either unity (UDF) or as those provided

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Fig. 1 Horizontal and vertical sections of the CC3 PWR benchmark problem.

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dimensions of each cell =  $1.4$  cm by 1.4 cm

Fig. 2 Heterogeneous PWR assembly geometry.

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Quarter Assembly	Elevation	Reference Relative Power	% Error ADF	% Error <b>UDF</b>	Quarter Assembly	Elevation	Reference Relative Power	% Error ADF	<b>&amp; Error</b> UDF
(i, j)	(k)				(i, j)	(k)			
1,4	$6$ (top) 5 4 3 $\overline{\mathbf{2}}$	0.3189 0.6458 0.9789 1.2705 1.1683	$-0.35$ $-0.50$ 0.00 0.28 0.40	$-2.57$ $-2.59$ $-1.65$ 0.51 1,00	2,4	6 5 4 3 $\overline{2}$	0.2614 0.5250 0.8271 1.5072 1.4276	$-2.33$ $-2.64$ $-2.03$ 0.19 0.31	0.77 0.69 1.31 0.39 0.52
		0.6698	0.51	1.11			0.8204	0.27	0.60
1,3	6 5 4 $\frac{3}{2}$ $\mathbf{1}$	0.4547 0.9138 1.3372 1.6108 1.4425 0.8235	$-0.66$ $-0.64$ $-0.24$ 0.01 0.24 0.36	$-1.85$ $-1.70$ $-0.91$ 0.01 0.45 0.68	2,3	6 5 4 3 $\mathbf 2$ $\mathbf{1}$	0.3514 0.7063 1.0431 1.3040 1.1784 0.6723	$-0.23$ $-0.32$ 0.09 0.23 0.39 0.48	$-2.16$ $-2.11$ $-1.35$ 0.54 0.90 1.09
1,2	6 5 4 $\frac{3}{2}$ $\overline{1}$	0.5186 1.0342 1.4669 1.6740 1.4603 0.8278	$-0.42$ $-0.31$ $-0.13$ 0.00 0.19 0.34	$-1.23$ $-0.99$ $-0.47$ 0.00 0.46 0.66	2, 2	6 5 4 3 $\boldsymbol{2}$ $\mathbf{1}$	0.4194 0.8373 1.1901 1.3651 1.1933 0.6758	$-0.22$ $-0.14$ 0.00 0.21 0.39 0.49	$-0.79$ $-0.56$ $-0.09$ 0.51 0.90 1.10
1,1	6 5 3 $\overline{\mathbf{c}}$ $\mathbf{1}$	0.4519 0.8980 1.2544 1.3986 1.2034 0.6782	$-0.13$ $-0.03$ 0.05 0.17 0.30 0.47	$-0.53$ $-0.28$ 0.05 0.46 0.80 1.09	2,1	6 5 4 3 $\overline{\mathbf{2}}$ $\mathbf{1}$	0.5525 1.0965 1.5324 1.7100 1.4720 0.8307	$-0.27$ $-0.14$ $-0.09$ 0.00 0.20 0.33	$-0.97$ $-0.68$ $-0.35$ 0.06 0.41 0.65

Table **1:** Nodal Pcwer Results for the Three-Dimensional **CC3** Problem

 $\sim 10^{11}$  km s  $^{-1}$ 

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 $\ddot{\phantom{a}}$ 

 $\sim$ 

by a 2D fine-mesh calculation run for the assembly with zero-current boundary conditions imposed (ADF). The ADF results are superior everywhere except in the rodded portion. This result is consistent with previous 2D results, which show that color set determinations of discontinuity factors are superior to assembly calculations for rodded nodes. In any event the use of unity factors for the axial direction seems satisfactory.

The BWR benchmark problem (TRD) is shown on Fig. **3,** the assembly geometry and assumed void distribution being shown on Figs. 4 and 5. Nodal powers (as edited from a fine-mesh, **3D,** PDQ calculation provided by Northeast Utilities) and QUANDRY errors arising from the use of UDF's and ADF's are given by Table 2. Unity axial discontinuity factors were used in both cases.

Again the ADF's provide superior results except in the rodded nodes and adjacent to the top reflector. Since 2D results have shown this same behavior, we conclude again that the use of unity discontinuity factors for the axial direction is a legitimate approximation.

Both the CC3 and TRD benchmarks involve reflecting boundary conditions in the radial plane. In order to determine whether the presence of a radial reflector invalidates the use of unity discontinuity factors for axial interfaces, we have analysed a small 3D benchmark problem which incorporates features of a PWR. The geometric characteristics of this benchmark (called the EPRI-9-3D problem) are shown on Fig. 6. The assemblies making up the reactor are of two enrichments (F-1 and F-2) and are geometrically those shown on Fig. 2. The sixteen control rod fingers (the shaded areas of Fig. 2, excluding the central one) are present in the shaded regions of Fig. 6.

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 $\Phi = 0$ 



Axial Core Layout for the TRD BVR Benchmark Problem Figure **3:**

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 $\ddot{\phantom{0}}$ 

Figure 4: Assembly Description for the TRD BWR Benchmark Problem

 $\ddot{\phantom{a}}$ 





REGION 1 REGION 2





**REGION 3 REGION** 4

Figure 5: Void Fractions for the TRD BWR Benchmark



PLn is the  $\begin{matrix} \texttt{th}\ \texttt{15cm}\ \texttt{plane}\ \texttt{grouping.}\ \texttt{n=1}\ \texttt{corresponds}\ \texttt{to}\ \texttt{the}\ \texttt{bottom} \end{matrix}$ fueled slice of the core.

Table 2: Errors in Nodal Powers for the TRD-BWR Benchmark.

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Fig. 6 The EPRI-9 3D Benchmark Problem.

 $\mathcal{L}_{\text{max}}$  and  $\mathcal{L}_{\text{max}}$ 

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 $\hat{\mathcal{A}}$ 

 $\epsilon$ 

We have already mentioned that, for rodded nodes, radial discontinuity factors based on 2D fine-mesh assembly calculations with zerocurrent boundary conditions lead to errors in nodal power of several percent. Figures 7, 8 and 9 illustrate this situation. Figure 7 shows comparisons of QUANDRY and PDQ for the 2D unrodded slice of the EPRI-9-3D problem. All homogenized group cross sections and discontinuity factors for the fuel nodes (4 nodes per assembly) were computed from assembly calculations (indicated by "A" on the lower figure). Color set calculations involving quarter assembly nodes and neighboring bafflereflector nodes were used to determine homogenized cross sections and discontinuity factors for the nodes on the reflector side of the core-baffle interface (indicated by "C3" on the lower figure). The errors in nodal power are quite acceptable  $($ 

Figure 8 shows the results when the same procedure is applied to a rodded radial slice of the 3D benchmark. The error of 2. 45% in power for the rodded node is probably acceptable, but it is much greater than that in the unrodded nodes.

Figure 9 illustrates what happens if color set calculations are used to determine homogenized cross sections and discontinuity factors for the rodded slice. Errors in nodal power predictions as compared with PDQ are again < 0. **5%.** Accordingly, in order to obscure as little as possible! errors due to use of unity-valued axial discontinuity factors,we have used for the 3D analysis radial parameters based on color set calculations for the rodded nodes.

On Fig. 10 the 3D QUANDRY predictions of nodal power are compared with those edited from a two-group, fine-mesh  $($   $\sim$  130,000 mesh point) PDQ, run for us by Northeast Utilities. The comparison is very

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Fig. 8 EPRI-9 Problem - 2D - (60 x 60 PDQ) with Control Rod.  $(8 \times 8$  QUANDRY) (Color Sets obtained with 1.4 cm mesh int.)



RELATIVE POWER DENSITY

 $(a)$  PDQ

(b) QUANDRY

(c)  $%$  Difference

مر 7  $C S$  $C<sub>5</sub>$  $c<sub>5</sub>$  $cs$ سر  $\mathsf{CS}\phantom{0}$  $CS$ A A ╱ A A  $\overline{\mathcal{L}}$  $CS$ Á.  $CS$ A A A  $\hat{\mathsf{F}}$ :Å  $\mathbf{A}$  $C$  S  $\Delta$  $5 \pm$  $\mathbf{A}$ A.  $\mathcal{L}_{\tau}$  $\boldsymbol{\mathsf{A}}$  $\overline{L}$ A A ا ک¢ A A A,  $c_{\mathcal{S}}$ A 斥  $\mathbf{A}$ A A A Assembly Calculation for JF & XS<br>Color Set Calculation for DF & XS **CS** 

Fig. 9 EPRI-9 Problem - 2D - (60 x 60 PDQ Reference Solution)<br>(8 x 8 QUANDRY)<br>(Color Sets obtained with 1.4 cm mesh int.)

 $\lambda_{\text{PDQ}} = 0.896628$  $\lambda_{\mathbf{Q}}$  $= 0.8962065$  $\Delta \lambda$  $= 0.047%$ 

 $\mathbf{I}$ 

RELATIVE POWER DENSITY

 $(a)$  PDQ

(b) QUANDRY

(c) % Difference



 $\epsilon$ 

 $\mathcal{A}$  $\mathbb{Z}^2$ 

 $(a)$ 

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Fig. 10 EPRI-9 Problem - 3D -  $(61 \times 61 \times 35 \text{ PDQ})$ Relative Power Density  $(8 \times 8$  QUANDRY)

- $(a)$  PDQ
- (b) QUANDRY
- (c) % Difference



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 $\overline{\phantom{a}}$ 

 $\overline{\phantom{a}}$ 



PLANE # 1

PLANE # 2

0350681

0.37832

 $0.67\%$ 

0.449983

0.45190  $-0.43%$   $\diagup$ 



PLANE  $# 3$  (RODDED)

 $-0,29\%$ 

 $0.613296$ 

060911

 $0.68%$ 

0.845655

0.84562

 $0.004$ %

1.10323 1.10 64

> $(RODOED)$ PLANE  $#4$

 $\lambda$ PDQ = 0.841384 }<br> $\lambda$ QUA = 0.841123 }  $\Delta\lambda = 0.031 \%$  encouraging. The difference in eigenvalue is **0.** 031%, and the maximum difference in nodal powers is **0. 71%.**

These differences are less than those which occur if the **PDQ** mesh size in the radial plane is cut in half. This fact is illustrated in Fig. **11** which shows the **2D PDQ** and **QUANDRY** results for the rodded plane when the radial mesh interval is taken to be **0.** 7 cm rather than the **1.** 4 cm used (for both PDQ and assembly-color-set calculations) to produce Figs. 7 to 10. Comparing the quarter-core PDQ results given in Figs. 9 and 11 shows that the difference in eigenvalue (due to mesh size alone) is **0.** 075%, and the corresponding maximum difference in nodal power is 4. 6%. Presumably, roughly the same difference would be observed if a finer mesh 3D PDQ were run. However, Northeast Utilities could not get this larger problem ( $\sim$  500,000 mesh points) to run on their IBM 3033 computer.

At present the preparation of the QUANDRY input must be done **by** hand; thus the manpower requirements to obtain the QUANDRY solution are substantial. Machine CPU time expended is, however, small: about 20 seconds to run the required fine-mesh assembly and color set PDQ's, another 5 seconds to obtain discontinuity factors from the output and 8 seconds to run the final 3D QUANDRY solution. The CPU time for the PDQ was **-** 97 minutes. (All times are on an IBM 3033.)

### IV. The Treatment of Partially Rodded Nodes

The test cases described in the previous section suggest that use of unity-valued discontinuity factors in the axial direction yields acceptably accurate results, provided there is a nodal interface at any location where there is a significant change in assembly characteristics, for example at the tip of a control rod. If a control rod is partially inserted in a node,

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 $(E4T1R)$ 

- (a) PDQ **-** Reference Solution (120 x 120)
- **(b) QUANDRY** Solution **8** x **3** Problem.
- (c) Relative Error in %.

 $\lambda_{\text{PDO}}$  = 0.897297  $\frac{1}{96}$  **Error** = 0.054%  $\lambda_{\mathbf{Q}}$  $= 0.8968136$ 

Shadowed area indicates where the control rod is inserted.

**AXS** and **ADF** have been obtained through color set calculations for nodes near the baffle and neighboring the control rod. Assembly calculations have been performed for the other inner nodes.

the radially homogenized cross sections for that node will change in a step fashion at the axial location of the tip of the rod, and it is quite unlikely that unity discontinuity factors at the axial faces of the node and simple volume weighted cross sections in its interior will lead to accurate predictions of reactivity and nodal power distribution.

It is of course possible to circumvent this difficulty **by** requiring that there be a nodal interface plane positioned at every control rod tip. But if one is dealing with part-length or stuck rods or is performing a search for the control rod bank position corresponding to the critical condition, this procedure can become complicated - particularly if powerdependent feedback effects (due to Xe, Sm, thermal-hydraulics, etc.) are being accounted for. For reactor transients during which control rods are moving, complications become even more severe. Thus there is considerable motivation for locating axial nodal interfaces at fixed planes and dealing with the intra-node axial discontinuities associated with partially inserted control rods **by** using some combination of non-unity axial discontinuity factors and/or axially flux weighted homogenized cross sections.

The September-April progress report showed that the use of discontinuity factors can get rid of the control rod cusping problem in QUANDRY computations. The possibility of making use of tabulation-interpolation methods was examined. However, the results of the investigation showed that these methods are problem-dependent, require complete global calculations, and demand considerable machine memory space - all unattractive features.

Since that time we have found a simple systematic, problemindependent way to solve the cusping problem in QUANDRY using data

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internal to the code. Results obtained using the new method have been compared with those of both the volume-weighted-cross section approximation method and the quadratic axial flux shape approximation currently programmed into QUANDRY. In view of the results described in the previous section, radially homogenized cross sections and discontinuity factors were assumed to have been found already, and reference solutions were finer mesh QUANDRY solutions.

For estimation of correct homogenized cross sections (XS) and axial discontinuity factors (DF) of a partially rodded node (PRN), an axial flux reconstruction method has been introduced. The purpose of the flux reconstruction is to predict (in a step called "Predictor") the axial flux shape within a PRN of the system of interest using information already available from the same system and collected in another step called "Collector". If predicted correctly, the XS and DF of a PRN thus obtained will yield the correct reactor solution for both static and transient situations.

Actually two such methods have been examined. The first is relatively simple but, for some situations, somewhat inaccurate. The second is more complicated but is capable of performing more accurate predictions adequate for transient calculations.

### Method 1

This method makes use of the nodal flux shape information implicit in the QUANDRY solution for the case of the control rod tip positioned exactly at an axial nodal interface. Two adjacent nodes, one fully rodded (r) and the other completely unrodded (u) (see Fig. 12(A) are solved for the one-dimensional axial flux shapes within them, with boundary currents on the axial faces and transverse leakages known from

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B. METHOD 2



Fig. 12 The Collector-Predictor Methods.

the global QUANDRY solution. Transverse leakage terms can be approximated as quadratic in shape. Then the axial flux shapes for the two nodes are displaced (see Fig. 12(A)) to construct an approximate flux shape within the partially rodded node (PRN). With this new flux shape, one can calculate FWC. These homogenized cross sections and the surface currents at the axial faces of the PRN region (lower portion of Fig. 12(A)) can be used to calculate  $\frac{1}{2}$ <sup>hom,</sup> surface Thus the discontinuity factors  $f \frac{1}{g}$  for the PRN can be found.

In addition to homogenized cross sections and axial discontinuity factors for the PRN, one needs to know radial discontinuity factors. Fortunately, without any supplemental calculation, the radial DF's can be estimated by making use of the volume-averaged fluxes  $\Phi_{\bf (h_c)}^{\bf P, \, PNN}$  and u, PRN for the two regions of a PRN (A' and B' in Fig. 12). Derivation of  $(h_{\alpha})$ this approximation is sketched on Fig. 13 for the surface x<sup>+</sup>. Many test results have shown that this approximation is acceptable (accuracy greater than **99.** 0% for most radial **D.** F. estimations of a PRN).

The basic assumption of this first method is that the axial flux shapes around the control rod tip do not vary significantly when the control rod travels the distance of one axial node.

### Method 2

To improve Method 1, we make use of the flux shape already constructed for a previous partially rodded condition. Thus the shape between the dotted lines in the upper portion of Fig. 12(B) is translated and assumed to be valid in the lower PRN region. Thus we solve (analytically) a one-dimensional, three-region problem for axial flux shapes throughout regions U, A, and B. Then we edit out the axial flux shape appropriate to the altered control rod tip position (lower portion of





# Fig. 13 Radial Discontinuity Factors for the Partially Rodded Node.

Fig. 12(B)). For a displacement shorter than the axial node size, this second method will predict more accurate homogenization parameters than Method 1. It is particularly useful for transient rod withdrawal problems.

The Collector-Predictor methods, along with the conventional volume weighted cross section (VWC) and quadratic axial flux approximation methods were tested and the results were compared for the CC3-PWR and TRD-BWR bench mark problems described in section III of this report. However, to avoid the errors discussed earlier arising from use of isolated assembly calculations to compute radially homogenized cross sections and discontinuity factors, two-dimensional calculations for full planes were done to obtain the radial AXS and DF's for nodes in different kinds of planes. Thus, errors due to the use of AXS and ADF from single, isolated assembly calculations were not present in the test problems.

For the CC3-PWR model (see Fig. 1 ) static tests were performed for the tip of the control rod located at 11 positions ranging from the 85 cm to the 138 cm axial location (see Fig. 1 ). As mentioned above, radially homogenized nodal cross sections and discontinuity factors were found from full planar, fine-mesh calculations with all water hole and control rod heterogeneities represented explicitly. Three-dimensional QUANDRY problems making use of the resultant AXS's and DF's, and with an extra axial plane inserted at the location of the tip of the control rod in the PRN, were taken as numerical standards (unity valued axial discontinuity factors were assumed for these standard calculations). Eigenvalues and power distributions as computed by Method 1 and by the conventional volume weighted cross section method (VWC-axial discontinuity factors unity and v v v PRN homogenized cross sections given by  $\Sigma^{\rm PKN}$  =  $\frac{\rm r}{\rm v}_{\rm r}$  +  $\rm v_{\rm r}$   $\Sigma^{\rm r}$  +  $\frac{\rm u}{\rm v}_{\rm r}$  +  $\frac{\rm u}{\rm v}_{\rm r}$   $\Sigma^{\rm U}$ were compared with these numerical standards. Results are displayed in

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Table **3** and Fig. 14. Clearly Method **1** is far superior to the VWC scheme. The rod cusping effect can be seen to be present for the VWC results but absent when Method 1 is used. The same CC3-PWR model was used for transient tests. The control rod was withdrawn for **0. 1** sec from the **100** cm to the 120 cm position. The withdrawal was accomplished in ten time steps. The results are shown in Table 4 and Figs. **15** and **16.** The predictions of both Method **1** and Method 2, along with VWC and the quadratic axial flux method programmed into **QUANDRY** (axial f's **= 1;** axial flux shape in the PRN a quadratic fit to the average **flux** levels in that node and its two axial neighbors), were compared with the standard, which used unity axial discontinuity factors but had a sufficient number of axial planes that the control rod tip was on an axial plane at each time step.

The results show that both Methods 1 and 2 are much more accurate than the VWC or quadratic axial flux schemes. Since the cost is only marginally greater, Method 2 (the most accurate) is favored.

A similar study was performed for the TRD-BWR model specified **by** Figs. **3** to **5.** Static results for the rod fixed at various locations in an internal axial node and in an axial node next to the reflector are displayed in Figs. **17** and **18.** Table **5** provides the numerical data. The acceptability of Method **1** as well as its superiority to the convensional VWC method is again demonstrated.

Figures **19** and 20 display the results of a rod withdrawal transient for the TRD-BWR. The withdrawal rate from the initial critical position was taken to be **300** cm/sec, and the transient was followed for **0.04** seconds. The inadequacy of the VWC model and the remarkable accuracy of the two new methods **-** particularly Method 2 **-** is again evident.

We believe that these results demonstrate that Method 2 provides a

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### Table 3

Comparison of keff and Max. Nodal Power Error of CC3-PWR Model (Static Problem)





Fig. 14 Comparisons of keff and Nodal Power for CC3-PWR.

# Table 4

CC3-PWR Transient Due to C.R. Withdrawal

(Total reactor power in watts, and % error.)



 $\Delta t = 10.0$  msec per 2 cm of rod withdrawal.



CONTROL ROD TIP POSITION

Fig. 15 Total Power Transition of CC3-PWR.



 $-20.$ 



Fig. 16 Total Power Error in Transition of CC3-PWR.

### Table 5

 $\mathcal{A}^{\text{max}}_{\text{max}}$ 

 $\bar{\epsilon}$ 

# Comparison of  $k_{eff}$  and Max. Nodal Power for Static TRD-BWR Problems



 $\sim$ 



 $-36-$ 

Fig. 17 Configuration and Comparisons of k<sub>eff</sub> and Nodal Power Error for Rod at Various Axial Positions in a Central Node.



Fig. 18 Configuration and Comparisons of keff and Nodal Power Error for Rod at Various Axial Positions in a Node Adjacent to the Bottom Reflector.





Fig. 19 Axial Layout and Numerical Results for the TRD-BWR Rod Withdrawal Transient.

 $\Box$ i



(rod moves 3 cm every 10 ms)

Fig. 20 Total Power vs Control Rod Position During Transient.

satisfactory resolution of the rod cusping problem for the QUANDRY code. The scheme can be applied simply and automatically entirely within the framework of the QUANDRY equations using data already generated during normal operation of the code. It is cheap to implement and appears to yield, consistently, very accurate results.

## V. Reconstruction of Fine Mesh 3D Flux Shapes from Nodal Solutions

In the September-April progress report we summarized tests of methods for reconstructing local fuel pin power from two-dimensional nodal solutions. The tests were for two-dimensional PWR and BWR benchmark problems depleted up to the beginning of a third fuel cycle with fuel shuffling and assembly rotation taking place between cycles. The basic assumption underlying this reconstruction is that the flux within a given heterogeneous node or on the nodal surfaces can be expressed as the product of a biquadratic polynomial (a "form function") multiplied by a fine-mesh "assembly" flux shape which reflects the detailed heterogeneous nature of the fuel assembly. For the PWR test case, assembly shapes using zero-current boundary conditions were sufficient to give maximum pin power errors of less than 5% for all the depletion steps. (Maximum error in the "hot" pin was 1. 02%.)

For BWR's this simple approach led to fuel assembly power errors as great as 5. 19%, and it was necessary to use an iterative response matrix approach to obtain accurate assembly powers and then an extended assembly scheme to obtain maximum fuel pin powers of under 5%.

This response matrix approach, while successful, is expensive (chiefly because of the cost of generating response matrices) and looks foreboding if extended to three dimensions. However, the PWR scheme is relatively straightforward and appears to be acceptably accurate. Accordingly we have extended it to the 3D-PWR benchmark shown in Fig. **6.**

Determining the quadratic form function from the nodal solution is necessary for both the PWR and BWR reconstruction procedures. Both require that corner point flux values be inferred from the nodal results. In two dimensions the assumption that the form function is quadratic on the mesh lines connecting corner points along with values of the average fluxes along these lines found from the nodal solution is sufficient to determine the corner point values. However, in **3D,** the line-averaged fluxes must themselves be inferred from the nodal solution. Accordingly, before presenting numerical results we shall sketch the overall theory of reconstructing detailed flux shapes using the output of nodal calculations along with fine-mesh fluxes found from zerocurrent assembly (or color set) calculations for the heterogeneous node of interest.

The basic assumption we make in order to perform a **3D** fine-mesh flux reconstruction is that for each group, **g,** and within any node (i, **j,** k) the reconstructed flux,  $\Phi_{\mathbf{R}_{_{\mathrm{cr}}}}^{\mathbf{ijk}}$  (x, y, z), can be expressed as the product of an assembly function,  $\psi_{\sigma}^{\bf lJ}(\bf x,y)$ , and a triquadratic polynomial function, g

$$
\Phi_{R_g}^{ijk} (x, y, z) = \psi_{g}^{ij} (x, y) \cdot P_{g}^{ijk} (x, y, z) \quad (g = 1, 2) \quad (1)
$$

where

$$
(x, y, z) \text{ is a point in node (ijk)};
$$
\n
$$
\Phi_R^{ijk} (x, y, z) = \text{unknown reconstructed flux};
$$
\n
$$
\psi_g^{ij} (x, y) = \text{known assembly flux};
$$

**pijk** (x, **y,** z): tri-quadratic polynomial in x, **y,** and z **g** with 27 unknown coefficients.

The assembly flux  $\psi_{\bf g}^{\bf 1J}$  (x, y) is obtained by performing a fine-mesh, 2D flux calculation (a PDQ zero-current assembly or color set calculation). The polynomial function can be written as:

$$
P_{g}^{ijk} (x, y, z) = \sum_{k=0}^{2} \sum_{m=0}^{2} \sum_{n=0}^{2} a_{g; k, m, n}^{ijk} x^{k} \cdot y^{m} \cdot z^{n} \quad (g = 1, 2)
$$
\n(2)

Note that separability is not assumed for this function. Thus there are 27 independent polynomial coefficients.

Once these "a" coefficients are found, and auxiliary color set calculations have been performed, one can compute the flux in any point within node (ijk). Thus the major problem consists of finding such polynomial coefficients. In order to find the polynomial coefficients it is first required that the reconstructed flux,  $\Phi_{\rm R}$ , when integrated over the nodal surfaces and the nodal volume, shall reproduce the nodal averaged values, which are obtained through a 3-D QUANDRY solution.

Thus, so far, we have a system of 27 unknowns (for each group and node) and a set of 7 equations (6 for the nodal surfaces and 1 for the node volume). To determine the "a" coefficients we need to find an additional set of 20 equations. This additional set of equations can be obtained by forcing the reconstructed flux  $\frac{\Phi^{ijk}}{\mathrm{Rg}}$  (x, y, z) to satisfy the 2-group Neutron Diffusion equations about nodal corner points, and by forcing the reconstructed flux to satisfy, in an integral sense, the 2-group Neutron Diffusion equations along nodal lines.

Thus we get one independent equation for each nodal corner point and one independent equation for each nodal line. Since there are 8 nodal corner points and 12 nodal lines, the number of balance equations per node is 20.

Thus we have (per group and node) a system of **27** equations: 7 obtained matching the nodal solution and 20 derived by satisfying the two-group neutron diffusion equations about corner points and along nodal lines. We can solve this system for the 27 unknown polynomial coefficients, and thus reconstruct the flux by using Equation (1).

By rewriting the polynomial function,  $\rho$ , appearing in Equation (2), and introducing a proper set of known "integral coefficients", which depend on the node geometry and assembly flux, it is possible to express this polynomial explicitly in terms of the **8** unknown corner point fluxes and 12 unknown line averaged fluxes, and in terms of the known surface averaged and volume averaged nodal values.

By doing this, we transform a general tri-quadratic polynomial, P, with **27** unknown coefficients into an equivalent polynomial, Q, with only 20 unknown coefficients (the surface averaged and volume averaged fluxes being known from the QUANDRY solution).

The reconstructed flux, expressed in terms of **Q,** becomes:

$$
\Phi_{\mathrm{Rg}}^{\mathrm{ijk}}\left(\mathbf{x},\mathbf{y},\mathbf{z}\right) = \psi_{\mathrm{g}}^{\mathrm{ij}}\left(\mathbf{x},\mathbf{y}\right) \cdot \mathrm{Q}_{\mathrm{g}}^{\mathrm{ijk}}\left(\mathbf{x},\mathbf{y},\mathbf{z}\right) \tag{3}
$$

where the coefficients of  $\mathsf{Q}^\textbf{ijk}_\mathbf{g}$  are such that  $\Phi^\textbf{ijk}_\mathbf{Rg}$  will reproduce the (still unknown) corner point fluxes,  $\Phi_c$ , and line averaged fluxes,  $\bar{\Phi}_L$  as well as the (known) face averaged and volume averaged nodal fluxes.

The polynomial function  $Q_g^{ijk}$  is an analytical function and thus continuous and differentiable within the node. However, the assembly function  $\psi_{\sigma}^{ij}$  is usually obtained by solving the two-group neutron diffusion equation **by** a finite difference approach. Thus it is neither continuous nor differentiable within the node.

In order to be consistent mathematically, it is convenient to treat the polynomial function as piecewise flat within mesh cubes, rather than as an analytical function, and then to solve the two-group finite difference neutron diffusion equations, rather than the two-group partial differential equations.

With this understanding we force the reconstructed flux to satisfy the two-group neutron balance equation about node corner points and along node lines in a finite different fashion.

First, the 3D reactor is divided into large nodes; then at any corner point **(ijk)** one can set the following system of equations:

1. Balance equation around corner point (i, j, **k).**

- 2. Balance equation along the node line parallel to x.
- 3. Balance equation along the node line parallel to y.
- 4. Balance equation along the node line parallel to z.

Since the reconstructed flux is expressed in terms of the unknown corner point fluxes and averaged fluxes along lines, then, the implementation of these equations at all node corner points will result in a system of coupled linear equations relating all corner point fluxes and averaged line fluxes. Since the coupling is very strong, and since both groups are coupled, one needs to solve the system of linear equations simultaneously by an iterative scheme.

In order to express a mesh point balance condition about corner point I, J, K we first note that, for mesh-point-centered difference equations (those used by PDQ), there will be a number of fine-mesh points between one corner point and the next, and the finite difference balance condition about that corner point will relate the group-g flux  $\Phi_{\infty}^{I,J,K}$  at that corner point to the group-g fluxes of its six nearest "fine-mesh"

neighbors and to the other group flux at (I, **J,** K). But the nearest "fine-mesh" neighbors all lie on lines connecting  $(I, J, K)$  to its six nearest "corner-point" neighbors  $I^{\pm}$ 1, J, K),  $(I, J^{\pm}1, K)$  and  $(I, J, K^{\pm}1)$ , and the flux shape along the line connecting, for example, corner point I, **J,** K to corner point  $I+1, J, K$  is being expressed as a product of the (known) assembly shape and a quadratic form function shape, which in turn is specified by  $\Phi_{cg}^{1, J, K}$ ,  $\Phi_{cg}^{1+1, J, K}$  and  $\Phi_{gx}^{1+1, J, K}$ , the average group-g flux along the line connecting  $I, J, K$  to  $I+1, J, K$ . Thus the "fine-mesh" flux nearest to I, J, K can be expressed as a linear function of  $\Phi_{\bf g}^{\bf I,J,K}$  $\bar{H}$ , J, K, and  $\bar{\Phi}^{l+1,\,J,\,K}_{gx}$ . Analogous expressions can be found for the other five "fine-mesh" fluxes nearest to  $I, J, K$ . Thus the finite difference balance condition for the mesh boxes surrounding a corner point connects I, J, K to its six nearest neighboring corner-point fluxes, the six line-**cg** averaged fluxes between I, J, K and the nearest "corner-point" neighbor, and the flux for the other group at I, J, K.

In an analogous fashion the unknown line-averaged flux  $\Phi_{\rm gx}^{\rm I+1,\,J,\,K}$ between I, J, K and I+1, J,K can be coupled to its four nearest parallel, line-averaged, "fine-mesh" fluxes, and these in turn (because of the assumed form of Equation (5)) can be expressed algebraically in terms of the four nearest (unknown) parallel "node-line-averaged fluxes" **+1,** J+-il, K **i+l, J,** Ktl SI+,J1K +1 **,** the (known) face-averaged fluxes on the four  $gx$   $gx$ planes connecting the node-line  $I, J, K$  - I+1,  $J, K$  to its nearest parallel neighboring node-lines and the ten corner point fluxes  $\Phi_{\text{gc}}^{I,J,K}$ ,  $\Phi_{\text{gc}}^{I+1,J,K}$ ,  $\overline{A}$ ,  $J^{\pm}$ <sub>1</sub>, K  $\overline{D}$ <sup>I+1</sup>, J<sup>+</sup>1, K  $\overline{D}$ <sup>I+1</sup>, J, K<sup>+1</sup> and  $\overline{D}$ <sup>I</sup>, J, K<sup>+</sup> gc **f**  $\int_{0}^{x}$   $\int_{0}$ 

Thus there are equations coupling each corner point flux and each line-averaged flux between corner points to other nearest neighbor corner point and line-averaged fluxes and to the (known) face averaged and volume averaged nodal fluxes.

**A** computer program solving for all these unknowns has been written and found to be very fast-running. We have applied it to finding corner point fluxes for the EPRI-9 3D benchmark problem shown in Fig. 6. Recall that the assemblies comprising this benchmark have the geometry shown in Fig. 2. It is the pin power at any location in these assemblies which we wish to reconstruct. The fluxes which give rise to that local power will be those specified by Eq. (3), where the  $\psi_{g}^{ij}(x, y)$ are known, two-dimensional fine-mesh shapes determined by assembly or color set calculations. Figures 8 and 9 show for the unrodded and rodded radial slices of Fig. 6 whether the  $\psi_{g}^{ij}(x, y)$  for particular nodes were taken from an assembly or a color set calculation. Recall that we use four nodes per assembly in the radial plane for these calculations. Also note that we have extended color set calculations into the bafflereflector regions, although we have not used the information generated by such calculations for the core region.

Since the "fine-structure" shapes  $\psi_{\stackrel{\ }{\bf g}}^{\bf 1J}({\bf x},{\bf y})$  of Eq. (3) are taken from two-dimensional assembly or color set calculations, all reconstructed spatial shapes in the z-direction must be accounted for by the tri-quadratic functions  $Q_g^{ijk}(x, y, z)$ . Thus at any fixed radial location (fixed x and y) the z-dependence of  $\Phi_{\rm Rg}^{\rm ijk}$  (x, y, z) will be represented as quadratic. However, because albedoe boundary conditions imposed at the axial surfaces of the reactor were taken to represent an infinite water reflector, the thermal flux inside the reactor near these axial surfaces falls off exponentially. As a result, representing the z-shape of the thermal flux as quadratic in the upper and lower 15 cm-sized nodes of Fig. 6 leads to severe  $($   $\sim$  19%) errors in reconstructed corner point fluxes in the axial surfaces of the reactor core. (Note, however, that since QUANDRY implicitly accounts

for the exponential behavior, the average nodal powers - see Fig. **10**  are in excellent agreement with the fine-mesh **3D PDQ** result.)

Using a zero-flux boundary condition at the core-reflector interface would probably get rid of this difficulty. However, since an albedoe boundary condition is more realistic and since the reference **3D PDQ** has been run subject to such a condition, we chose to overcome the difficulty **by** adding extra nodes, two-centimeters thick, on the core-side of the axial core-reflector interfaces. Thus for reconstruction purposes we reran **QUANDRY** maintaining the radial mesh layouts of Figs. **7** and **9,** but with the axial mesh spacings shown on Fig. 21. Assuming that the axial mesh shape can be represented **by** a quadratic in the 2 cm intervals next to the top and bottom reflectors and then **by** another quadratic in the **<sup>13</sup>**cm thick adjacent mesh intervals is far more accurate than assuming a single quadratic shape throughout the entire **15** cm intervals.

Previous calculations for two-dimensional cases have shown that the maximum error in the detailed, heterogeneous fluxes reconstructed throughout a node according to the two-dimensional counterpoint of **Eq. (3)** always occurs at the corner points. Accordingly, a good measure of the accuracy of our reconstruction procedure are the errors in predicted fluxes incurred at corner points. These errors are shown for the **3D** EPRI-9 problem in Figs. 22 and **23.** As can be seen, the maximum errors  $(s, 7, 5\%)$  occur on the radial periphery of the top and bottom planes where the fluxes are quite low. For all interior corner points (including those on the top and bottom axial interfaces) errors are less than one per cent. At the corner points of the highest-power node (the central node between planes 4 and **51** the corner point fluxes match the fine mesh PDQ results to better than **0. 3%.** Such agreement with a full

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Fig. 21 Axial View - 3D EPRI-9 Problem.



Fig. 22 Errors in reconstructed corner point fluxes; upper 4 mesh planes.



Fig. 23 Errors in corner point fluxes; lower 3 mesh planes.

core fine mesh **PDQ,** obtained using only fine mesh, **2D,** assemblysized calculations, and with no normalization even to **2D** quarter core results, is very satisfying.

# **VI.** The Use of Albedoe Boundary Conditions for Nodal Methods

The most straightforward way to represent the effects of a reflector on criticality and power distribution is to treat the reflector explicitly as a separate region. With two-group nodal codes such as **QUANDRY** this is the usual approach. However, for PWR's the relatively thin baffle region between the core and the water part of the reflector leads to homogenization problems which require use of color sets to cure. In addition, explicit representation of reflector regions increases the running time of nodal calculations **by** 20-30%, and, although, except possibly for three-dimensional transient problems, this does not increase significantly the total dollar cost of running a calculation, it is an extra expenses which would be worth avoiding if one could do so without serious penalty. Accordingly, we have for some time been searching for accurate and inexpensive ways to represent reflector effects **by** imposing a boundary condition at the core-reflector interface.

уk.

Previous work **by** Kalambokas dealt with the replacement of reflector and baffle nodes **by** albedoe boundary conditions in the context of finite difference methods (here the dollar amount of the saving can be significant). Analytical formulas for the prediction of such albedoes were developed and applied to various reactor problems. The purpose of this present work was to investigate the replacement of reflectors and baffles **by** albedoe boundary conditions in nodal calculations.

Computational nodes representing reflectors and baffles may be replaced **by** albedoe boundary conditions of the form:

$$
\begin{bmatrix} \Phi_1(u) \\ \Phi_2(u) \end{bmatrix} = \begin{bmatrix} \alpha_{11}(u) & 0 \\ \alpha_{21}(u) & \alpha_{22}(u) \end{bmatrix} \begin{bmatrix} J_1(u) \\ J_2(u) \end{bmatrix}
$$
 (4)

where the reactor core boundary is assumed to be at u. For onedimensional, one-region reflectors, exact analytical formulas can be derived. Specifically, the albedoes required to replace the reflector are:

$$
\alpha_{gg} = \frac{L_g}{D_g} T \text{ANH} \left( \frac{\Delta}{L_g} \right) \qquad g = 1,2 \qquad (5)
$$

$$
\alpha_{21} = \frac{\Sigma_{21}}{D_1 \Sigma_2 - D_2 \Sigma_1} (D_1 \alpha_{11} - D_2 \alpha_{22})
$$
 (6)

where

 $\Delta$  = the reflector thickness

 $L_g$  = the reflector diffusion length for group-g.

Multi-region extensions of Equations **(5)** and **(6)** have been derived, and we have generalized the **1D** analytical expressions to allow arbitrary outer surface boundary conditions. This generalization is of importance

for "thin" (i. e.  $\Delta$ <2 L<sub>g</sub>) reflector or baffle configurations.

The approximate analytical albedoe expressions for two-dimensional, one-region reflectors developed **by** Kalanbokas are:

$$
\alpha_{gg}(u) \approx \frac{L_g}{D_g} (1 + \exp{(\frac{-u}{L_g})}) \qquad g = 1, 2 \qquad (7)
$$

$$
\alpha_{21}(u) \approx \frac{r}{D_1} (L_1(1 + \exp{(\frac{-u}{L_1}))} - L_2(1 + \exp{(\frac{-u}{L_2}))})
$$
(8)

where

 $\mathbf r$ 

$$
= \frac{\sum_{21}}{D_2 \frac{1}{L_2} - \frac{1}{L_2}} \big)
$$

**<sup>u</sup>**= the distance along the reflector face from the corner. The analytical two-dimensional albedoes reduce to the one-dimensional albedoes for the case of an infinite reflector and  $u \gg L_{\sigma}$ . Unfortunately, we have shown **by** numerical studies that the approximate nature of the **2D** analytical albedoes and their limitation to one-region reflectors precludes the general use of analytical albedoe conditions at the core interface.

Local fine-mesh finite-difference calculations were then investigated as a method of determining albedoes at the core interface. First, fixed source calculations were performed for specially chosen subdivisions of the reflector. The objective was to generate response matrices from which the albedoe boundary conditions could be calculated. However, the high cost of response matrix generation and the inaccuracy of the fixed source calculations made this procedure very inefficient. Peripheral assembly color set calculations for PWR's were likewise found to be insufficient for interface core-albedo prediction. The primary difficulty in each case was the modelling of the side boundary conditions at the interfaces between adjacent reflector nodes. Imposing  $Jg \cdot n = 0$  side boundary conditions makes the local calculations economically feasible. However, the resultant fine-mesh flux shapes are not accurate enough for the albedoes to be estimated without large errors. The errors are large enough that a nodal calculation performed with them would have nodal power errors on the order of 15%.

The effort to calculate core-reflector interface albedoes without recourse to a reference solution was therefore abandoned. Nevertheless, two specific applications of the analytical albedoes have been developed and are discussed below.

First, it has been found that the analytical albedoes may be used to replace the outermost reflector nodes. Although the savings in problem size reduction are not as great as they would be for core-surface albedoes, the accuracy of nodal replacement is significantly enhanced.

The notational scheme used to describe problem size reduction by albedoe boundary conditions is illustrated in Fig. 24 for the LSH benchmark problems (a two-dimensional BWR problem composed of Vermont Yankee assemblies). A two-stage reduction corresponds to the case of one reflector node all around the core while a four-stage reduction corresponds to a bare core.

The fact that the 2D analytical albedoe formulas are restricted to homogeneous reflectors is not a restriction when albedoe boundary conditions are not imposed at the core-reflector interface (where in PWR's there is generally a baffle). In addition, the reactor is less sensitive to albedoe errors when the boundary conditions are moved away from the core.

The accuracy of replacing full reflectors by partial reflectors augmented by analytical albedoe boundary conditions can be tested by comparing nodal solutions for the truncated reactors to the original nodal solutions. This has been done for the LSH BWR, the EPRI-9 PWR, and

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1 Stage Reduction





**3** Stage Reduction

4 Stage Reduction (Bare Core)



the LRA BWR benchmark problems illustrated in Fig. 25. The results obtained from this comparison of nodal solutions are given in Table **6.** The background error for each problem is determined **by** comparison of the original nodal solution with a fine-mesh reference solution. Note that the LRA benchmark is composed of homogeneous nodes. Hence the background error is due only to the quadratic transverse leakage approximation inherent in **QUANDRY.** The **E9** benchmark is composed of the (rod-free) heterogeneous nodes shown in Fig. **26.** Discontinuity factors based on color set calculations for all interior quarter assemblies as well as the quarter-assembly-sized baffle-reflector nodes were used in the QUANDRY calculations. Thus nodal power predictions are good (< **0.** 5% error). The **LSH** benchmark, composed of Vermont Yankee assemblies (some of which are rodded), was analysed using only  $\mathbf{J}_{\rho} \cdot \mathbf{n} = 0$ assembly calculations to determine discontinuity factors. Hence (as with the TRD benchmark - Table 2) the maximum assembly power error (here '10%) is large. The other errors given in Table **6** are the additional nodal solution errors caused **by** problem size reduction. It is seen that a one or two stage reduction by analytic albedoes (the columns ANAL) causes virtually no additional error in the nodal solutions. The fact that such small errors are not obtained when a zero-incoming-partial-current boundary condition is imposed (the columns  $\alpha_{\text{gg}}^{\text{}}$  = 2) is evidence that the reflector is not effectively infinite.

The computational savings realized by 1 or 2 stage reductions is roughly proportional to the number of nodes which are truncated. Accordingly, a 1-stage reduction corresponds to a 10-15% savings while a 2-stage reduction corresponds to a 15-25% savings. Problem size

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Fig. 25 The LRA BWR Problem.

 $\hat{\mathbf{v}}$ 

# TABLE 6

# USE OF ALBEDO BC's FOR NODAL PROBLEM SIZE REDUCTION





Fig. 26 Peripheral Color Set Geomatries Superimposed on the EPRI-9 PWR Benchmark.

reduction by albedoe boundary conditions therefore leads to a modest gain in calculational efficiency with negligible error. Three-stage reduction by analytical albedoes did not prove to be acceptable. Errors much larger than the 2-stage  $\alpha_{\rm gg}^{\phantom{\dag}}$  = 2 results in Table 6 were obtained.

The errors given on Table 7 for the EPRI-9 benchmark show that cornerpoint flux interpolation from nodal results is not adversely affected by the replacement of the outermost reflector nodes by albedo boundary conditions. For comparison the (unacceptable) cornerpoint interpolation results from a 3-stage reduction are also given.

A second specific application of the analytical albedo boundary conditions which has been developed is an improved method for PWR baffle-homogenization. The usual method of baffle-homogenization makes use of peripheral assembly color set eigenvalue calculations to generate homogenized cross sections and discontinuity factors for the quarterassembly-sized nodes containing the baffle. The EPRI-9 peripheral assembly color set geometries are shown as the dotted squares in Fig. 26. The color set calculations were carried out with  $\frac{J}{\sigma} \cdot \underline{n}$  = 0 boundary conditions.

The improvement which has been made in this procedure is to replace the outer face boundary conditions of the A and B color sets with analytic albedoes. This change results in a more realistic color set flux shape and therefore in better homogenized cross sections and discontinuity factors. The changes are especially noticeable in the baffle nodes.

The results given in Table 8 show the improvement in the nodal solution which results if the modified peripheral assembly color sets are used. It would appear that about half of the error in the original EPRI-9 nodal solution was due to improper baffle homogenization.

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TABLE 7 CORNER POINT INTERPOLATION ON EPRI-9



(REFERENCE CORNERPOINT FLUKES OBTAINED FROM FINE MESH REFERENCE SOLUTIONS)



 $\mathcal{L}^{\text{max}}_{\text{max}}$  and  $\mathcal{L}^{\text{max}}_{\text{max}}$ 

### **VII.** Conclusions

We believe that our test results show that, at least for PWR's, the **QUANDRY** method should be implemented in utility code packages as a replacement for current nodal schemes. It is far more accurate than the present methods and achieves that accuracy without fitting any albedoes or adjustable constants to quarter core **PDQ** results. Its computer storage requirements are large. However, a new iterative scheme suggested **by** Kord Smith would reduce them **by** a factor of **3** or 4 so that there should be no problem with handling realistic problems within the fast memory on IBM equipment (and possibly **CDC** equipment). With two-groups represented explicitly, problem running time should be about twice that of the conventional nodal codes. However, the extra cost should be balanced against the savings which result from not having to run any quarter core PDQ's. Another advantage is that the method can be extended directly to the analysis of transient problems.

Moreover there are still areas for potential improvement. It may be possible to replace the four radial discontinuity factors per node per group **by** one average value, thereby reducing storage requirements. It may be possible to use discontinuity factors to reduce the number of groups from two to one. Also it may be possible to derive the **QUANDRY** parameters directly from sophisticated spectrum codes such as **CASMO,** thereby avoiding completely the use of **PDQ.** We hope to investigate these ideas during the coming year.

For BWR's we feel that more work is needed. Our test cases have shown that using discontinuity factors based on 2D,  $J_g \cdot n = 0$  assembly calculations leads to a considerable improvement in the accuracy of predicted nodal power over the standard nodal schemes (specifically

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SIMULATE). However, for rodded nodes, errors can still be as large as 10%, and reconstructed pin power results are even worse. We have overcome these difficulties **by** using iterative response matrix procedures. However, the cost of determining the response matrices is high. We hope to investigate methods for reducing that cost or to avoid it altogether **by** improving our noniterative schemes for finding assembly discontinuity factors. The use of color sets is one obvious possibility which should be explored.